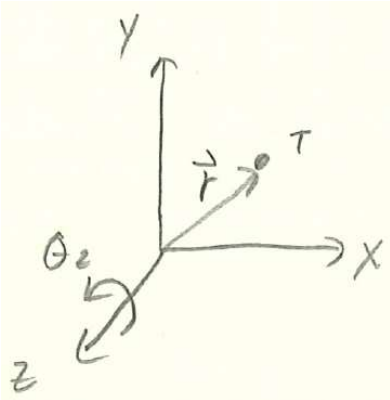


Inertial Sensors (MEMS Gyroscopes)

1) Let's examine the Coriolis force and acceleration

Consider:



Where the position vector is: $\vec{r} = r_x\hat{i} + r_y\hat{j}$, and

the velocity vector is: $\vec{V}_r = V_{rx}\hat{i} + V_{ry}\hat{j} = \frac{d}{dt}\vec{r}$.

Given that the coordinate system rotates about the z-axis at:

$\vec{\Omega}_z = \Omega_z\hat{k} = \dot{\theta}_z\hat{k}$, the object, T, experiences a “virtual force” in the x-y plane due the $\Omega_z\hat{k}$ rotation.

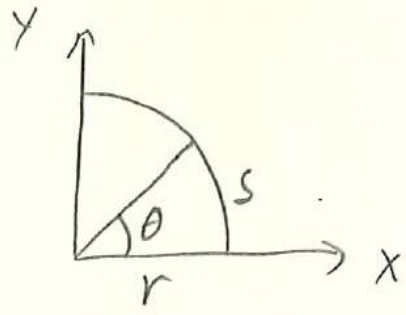
This “virtual force” is the Coriolis force, and it results in a Coriolis acceleration in the rotating x-y plane $\rightarrow \vec{a}_c$.

$$\vec{a}_c = 2\vec{\Omega}_z \times \vec{V}_r = \Omega_z\hat{k} \times (V_{rx}\hat{i} + V_{ry}\hat{j}) = 2\Omega_z V_{rx}\hat{j} - 2\Omega_z V_{ry}\hat{i},$$

where: $a_{cy} = 2\Omega_z V_{rx}$ and $a_{cx} = -2\Omega_z V_{ry}$.

Observe that $a_{cy} \propto \Omega_z$ multiplied by V_{rx} .

Another way to look at this:



$$S = r\theta$$

$$\dot{S} = \dot{r}\theta + r\dot{\theta}$$

$$\ddot{S} = \dot{r}\ddot{\theta} + \ddot{r}\theta + r\ddot{\theta} + \dot{r}\dot{\theta}$$

$$= 2\dot{r}\dot{\theta} + \ddot{r}\theta + r\ddot{\theta}$$

$$= 2\dot{r}\Omega_z + \ddot{r}\theta + r\alpha_z$$

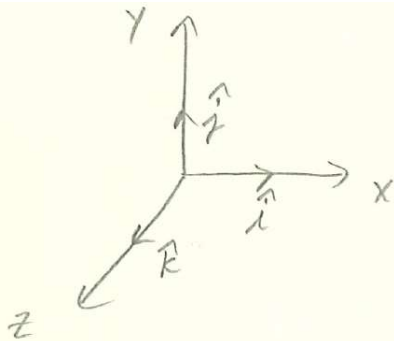
If $\theta \approx 0$ and $\ddot{\theta} \approx 0$, then $\ddot{S} \approx a_{cy} = 2V_{rx}\Omega_z$

If we were technicians, then knowing the \vec{a}_c equation would be sufficient. But we are engineers and we should therefore know more.

2) Review of unit vectors

A unit vector is a normalized vector of length 1.

In Cartesian co-ordinates: 3 unit vectors in the x, y, and z directions:



For the cross product of unit vectors: use the right hand rule:

$$(1) \quad \hat{i} \times \hat{i} = \hat{j} \times \hat{j} = \hat{k} \times \hat{k} = 0$$

$$(2) \quad \hat{i} \times \hat{j} = \hat{k}$$

$$\hat{k} \times \hat{i} = \hat{j}$$

$$\hat{j} \times \hat{k} = \hat{i}$$

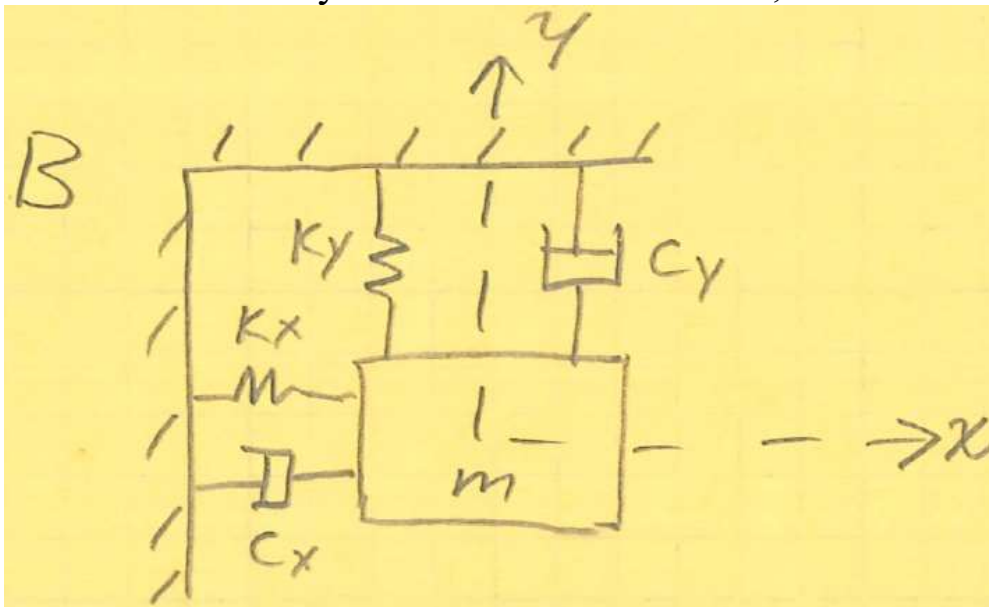
$$(3) \quad \hat{j} \times \hat{i} = -\hat{k}$$

$$\hat{k} \times \hat{j} = -\hat{i}$$

$$\hat{i} \times \hat{k} = -\hat{j}$$

3) Modeling an SMD in an inertial reference frame

Consider a SMD system in a reference frame, B:



Assume that the mass, m , can only move in the x and y directions with no rotation in B .

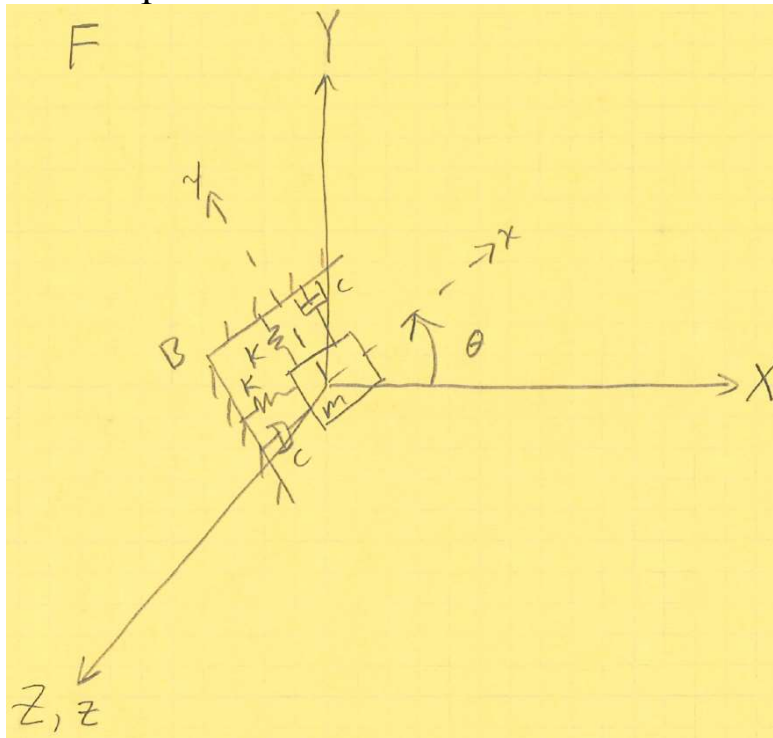
Although k_x is not necessarily the same as k_y , let's let $k_x = k_y = k$ for simplicity.

Therefore $\omega_{nx} = \omega_{ny}$ here.

Similarly, c_x is not necessarily the same as c_y , but let's let $c_x = c_y = c$ for simplicity.

4) Modeling an SMD in a rotating inertial reference frame

Let reference frame B be in a fixed reference frame F where B can rotate with respect to F :



X, Y, Z are in $F \rightarrow \hat{I}, \hat{J}, \hat{K}$: unit vectors in F .

x, y, z are in $B \rightarrow \hat{i}, \hat{j}, \hat{k}$: unit vectors in B .

Note: \hat{z} and \hat{Z} always point in the same direction.

Let's explore the relationship between \mathbf{B} and \mathbf{F} :

$$\hat{i} = f(\hat{I}, \hat{j}) \text{ and } \hat{j} = f(\hat{I}, \hat{j})$$

$$\therefore \hat{i} = \hat{I} \cos(\theta) + \hat{j} \sin(\theta)$$

$$\text{Note: if } \theta = 0^\circ \rightarrow \hat{i} = \hat{I}$$

$$\text{if } \theta = 90^\circ \rightarrow \hat{i} = \hat{j}$$

$$\therefore \hat{j} = -\hat{I} \sin(\theta) + \hat{j} \cos(\theta)$$

$$\text{Note: if } \theta = 0^\circ \rightarrow \hat{j} = \hat{j}$$

$$\text{if } \theta = 90^\circ \rightarrow \hat{j} = -\hat{I}$$

$$\text{Angular rate: } \dot{\theta} = \frac{d\theta}{dt} = \Omega$$

$$\text{Angular acceleration: } \ddot{\theta} = \frac{d\Omega}{dt} = \alpha$$

5) Derivatives of unit vectors

$$\text{Note: } \frac{d}{dt}(\cos(\theta)) = -\dot{\theta} \sin(\theta) = -\Omega \sin(\theta)$$

$$\text{And: } \frac{d}{dt}(\sin(\theta)) = \dot{\theta} \cos(\theta) = \Omega \cos(\theta)$$

With that in mind:

$$\begin{aligned} \therefore \frac{d}{dt}(\hat{i}) &= \frac{d}{dt}(\hat{I}\cos(\theta) + \hat{J}\sin(\theta)) \\ &= -\hat{I}\Omega\sin(\theta) + \hat{J}\Omega\cos(\theta) \\ &= \Omega(-\hat{I}\sin(\theta) + \hat{J}\cos(\theta)) \\ \text{But: } \hat{j} &= -\hat{I}\sin(\theta) + \hat{J}\cos(\theta) \\ \therefore \frac{d}{dt}(\hat{i}) &= \Omega\hat{j} \end{aligned}$$

$$\begin{aligned} \therefore \frac{d}{dt}(\hat{j}) &= \frac{d}{dt}(-\hat{I}\sin(\theta) + \hat{J}\cos(\theta)) \\ &= -\hat{I}\Omega\cos(\theta) - \hat{J}\Omega\sin(\theta) \\ &= -\Omega(\hat{I}\cos(\theta) + \hat{J}\sin(\theta)) \\ &= -\Omega\hat{i} \end{aligned}$$

Therefore: $\dot{\hat{i}} = \Omega\hat{j}$ and $\dot{\hat{j}} = -\Omega\hat{i}$

Identities

$$\begin{aligned} \hat{i} &= \hat{I}\cos(\theta) + \hat{J}\sin(\theta) \\ \hat{j} &= -\hat{I}\sin(\theta) + \hat{J}\cos(\theta) \\ \dot{\hat{i}} &= \Omega\hat{j} \\ \dot{\hat{j}} &= -\Omega\hat{i} \end{aligned}$$

Corolis acceleration: \vec{a}_c

where $\vec{a}_c = 2\Omega\dot{x}\hat{j} - 2\Omega\dot{y}\hat{i}$

→ motion in one axis (\dot{x} or \dot{y}) plus rotation about z (Ω) results in motion in the opposite x or y axis:

$$\dot{x}\hat{i} \text{ and } \Omega\hat{k} \rightarrow 2\Omega\dot{x}\hat{j}$$

$$\dot{y}\hat{j} \text{ and } \Omega\hat{k} \rightarrow -2\Omega\dot{y}\hat{i}$$

However, higher order terms exist.

Consider the motion of the proof mass:

$$\vec{r} = x\hat{i} + y\hat{j} \quad \text{displacement of m}$$

$$\begin{aligned} \vec{v} = \dot{\vec{r}} &= \dot{x}\hat{i} + \dot{y}\hat{j} + x\dot{\hat{i}} + y\dot{\hat{j}} \\ &= \dot{x}\hat{i} + \dot{y}\hat{j} + \Omega(x\hat{j} - y\hat{i}) \quad \text{velocity of m} \end{aligned}$$

$$\begin{aligned} \vec{a} = \dot{\vec{v}} &= \ddot{x}\hat{i} + \ddot{y}\hat{j} + \dot{x}\dot{\hat{i}} + \dot{y}\dot{\hat{j}} + \dot{\Omega}(x\hat{j} - y\hat{i}) + \Omega(\dot{x}\hat{j} - \dot{y}\hat{i}) + \Omega(x\dot{\hat{j}} - y\dot{\hat{i}}) \\ &= \ddot{x}\hat{i} + \ddot{y}\hat{j} + \Omega(\dot{x}\hat{j} - \dot{y}\hat{i}) + \alpha(x\hat{j} - y\hat{i}) + \Omega(\dot{x}\hat{j} - \dot{y}\hat{i}) + \Omega^2(-x\hat{i} - y\hat{j}) \\ &= \ddot{x}\hat{i} + \ddot{y}\hat{j} + 2\Omega(\dot{x}\hat{j} - \dot{y}\hat{i}) + \alpha(x\hat{j} - y\hat{i}) - \Omega^2(x\hat{i} + y\hat{j}) \quad \text{acceleration} \\ &\text{of m} \end{aligned}$$

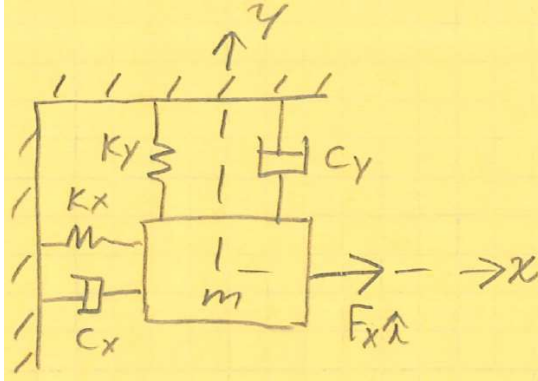
From this expression for \vec{a} :

$$a_x = \ddot{x} - \alpha y - 2\Omega\dot{y} - \Omega^2 x \quad \text{Acceleration component along x}$$

$$a_y = \ddot{y} + \alpha x + 2\Omega\dot{x} - \Omega^2 y \quad \text{Acceleration component along y}$$

6) System dynamics

Consider this model for the MEMS SMD mechanical system:



$F_x = A_x \sin(\omega_d t) \rightarrow$ to force m to oscillate along x -axis (using an actuator)

$F_y = 0 \rightarrow$ no force applied to m along y -axis (with an actuator)

There exists a coupling of the equations of motion:

$$ma_x + c_x \dot{x} + k_x x = F_x \quad (1)$$

$$ma_y + c_y \dot{y} + k_y y = F_y = 0 \quad (2)$$

Expanding these equations:

$$m(\ddot{x} - \alpha y - 2\Omega \dot{y} - \Omega^2 x) + c_x \dot{x} + k_x x = A_x \sin(\omega_d t) \quad (1)$$

$$m(\ddot{y} + \alpha x + 2\Omega \dot{x} - \Omega^2 y) + c_y \dot{y} + k_y y = 0 \quad (2)$$

We want to solve this set of equations to obtain an expression for $y(t)$. Thankfully, we can make some reasonable simplifying assumptions:

- (1) Let $k_x = k_y = k$
- (2) Let $c_x = c_y = c$

Note: with (1) and (2): $\omega_{nx} = \omega_{ny} = \omega_n$. Real MEMS gyroscopes usually have $\omega_s > \omega_d$: defined as $\omega_{ny} > \omega_{nx}$, where ω_s is in regard to the sense side and ω_d is in regard to the drive side. Having $\omega_s > \omega_d$ yields better stability and a measurable rotation rate bandwidth.

- (3) Assume that the angular acceleration, α , is very slow and can be approximated as $\alpha = 0 \text{ rad/s}^2$.
- (4) Assume that the system natural frequency, ω_n , is much greater than Ω , the angular rate being measured. Therefore $\Omega^2 x$ and $\Omega^2 y$ can be approximated by 0.

Example: if $f_n = 10 \text{ kHz}$: $\omega_n = 2\pi f_n = 62,831.8 \text{ rad/s}$
 If $\Omega = 300 \text{ }^\circ/\text{s} = 300(2\pi/360) = 5.24 \text{ rad/s}$
 And $62,831.8 \gg 5.24$

Also from EQ 1: $m(\ddot{x} - \alpha y - 2\Omega\dot{y} - \Omega^2 x) + c_x\dot{x} + k_x x = A_x \sin(\omega_d t)$

Examine the “x” terms: $-m\Omega^2 x + k_x x \rightarrow m\left(\frac{k_x}{m} - \Omega^2\right) = m(\omega_n^2 - \Omega^2)$

From the Ω and f_n terms above $\rightarrow \omega_n^2 = 3.9 \times 10^9 \text{ rad/s}$ and $\Omega^2 = 27.5 \text{ rad/s}$. So, $\omega_n^2 - \Omega^2 \approx \omega_n^2$

- (5) The amplitude of the motion of m along the x-axis will be tightly controlled as a closed loop resonator.

A feedback control system will adjust $F_x = A_x \sin(\omega_d t)$ to precisely keep the motion along the x-axis exactly as desired. Therefore, we can drop the $a_{cx} = -2\Omega\dot{y}$ term in EQ 1, since the controller will null out its effect.

(6) The ω_d from $F_x = A_x \sin(\omega_d t)$ is usually selected so that:

$$\omega_d = \omega_n = \sqrt{\frac{k}{m}}.$$

This minimizes the amplitude of F_x required to achieve sufficient motion of m along the x -axis \rightarrow due to high Q .

Therefore, the equations of motion simplify to:

$$m\ddot{x} + c\dot{x} + kx = A_x \sin(\omega_n t) \quad (1)$$

$$m\ddot{y} + c\dot{y} + ky + 2m\Omega\dot{x} = 0 \quad (2)$$

Clearly, $\Omega\hat{k}$ and motion along the x -axis produces corresponding motion along the y -axis {useful if \dot{x} is consistent (periodic and known)}.